

ChE 344

Reaction Engineering and Design

Lecture 15: Tuesday, Mar 8, 2022

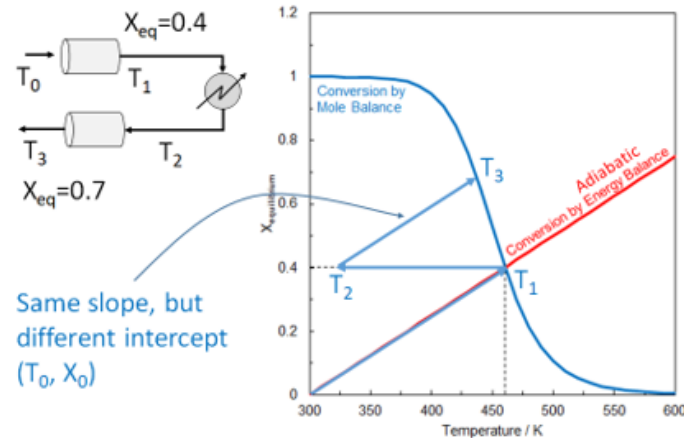
Non-isothermal reactor design: Reactor staging,
Heat exchange

Reading for today's Lecture: Chapter 11.6, 12.1

Reading for Lecture 16 : Chapter 12.1, 12.2, 12.4

Interstage heat exchangers

For an exothermic reaction, $A \rightleftharpoons B$ we can view the equilibrium conversion vs. the conversion from the adiabatic energy balance. The intercept (T_1) is the maximum conversion you can achieve in a single adiabatic reactor. However, by cooling the reactants/products, you can get a higher conversion in a second reactor.

For a heat exchanger coupled with a reactor (PFR)

From PFR energy balance

$$\frac{dT}{dV} = \frac{r_A \Delta H_{rxn} - Ua(T - T_a)}{\sum F_i C_{p,i}} = \frac{r_A \Delta H_{rxn} - Ua(T - T_a)}{F_{A0} [\sum \theta_i C_{p,i} + \Delta C_p X]} = \frac{Q_g - Q_r}{\sum F_i C_{p,i}}$$

T_a is the temperature of the heat exchanger fluid at volume V .

Q_g is the heat 'generated' by the reaction, and Q_r is the heat 'removed' by the heat exchanger (positive if $T > T_a$, negative if $T < T_a$).

U is the heat transfer coefficient with units of $J/m^2 \cdot s \cdot K$

a is the shape factor, or heat transfer area divided by unit reactor volume.

From PFR mole balance

$$F_{A0} \frac{dX}{dV} = -r_A$$

Can solve this coupled set of equations for PFR design.

Progress in expanding our reactor repertoire

For Midterm 1, **isothermal** CSTRs, PFRs/PBRs, batch reactors, isobaric flow with a single reaction.

Isobaric flow reactors:

$$P = P_0 \quad \text{or more rigorously} \quad \frac{dP}{dV} = 0 \quad \frac{dP}{dW} = 0$$

Then we introduced **pressure drop** for packed bed reactors which we modeled using the Ergun equation (for gases here).

$$\frac{dp}{dW} = - \frac{\alpha}{2} \frac{1}{p} \frac{T}{T_0} \frac{F_T}{F_{T0}}$$

We needed to solve a coupled set of equations when adding the Ergun equation because this pressure drop influenced our volumes for gases, and thus concentrations.

Next we added to our menu a new set of reactors

Semi-batch (Lecture 9) and membrane reactors (Lecture 10)

We also added the concept of using experimental data to determine rate laws (Lecture 11) and multiple reactions (Lecture 12).

The most recent step was to study the effect if the reactors are not isothermal (Lecture 13-14)

Isothermal reactors:

$$T = T_0 \quad \text{or more rigorously} \quad \frac{dT}{dV} = 0$$

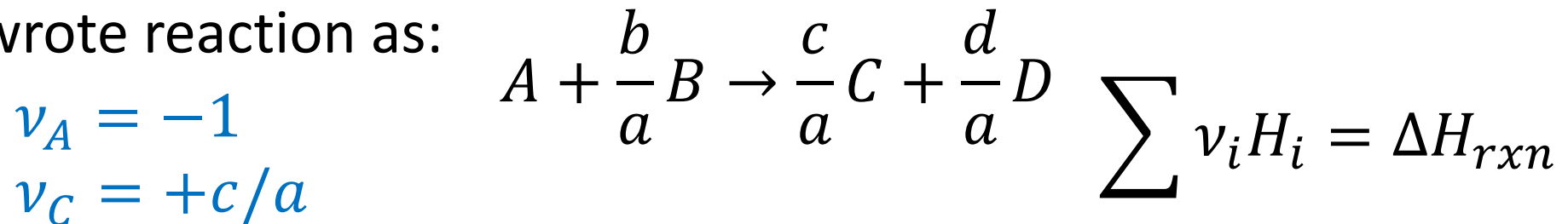
Many of the reactions that are run in reality may be exothermic or endothermic, which we know can change the temperature of the reactants/products.

For our adiabatic reactors from the energy balance, assuming

- No heat in/out
- No shaft work
- Steady state

$$X_{adiabatic} = \frac{\sum \theta_i C_{P,i} [T - T_0]}{-[\Delta H_{rxn}(T_{ref}) + \Delta C_P (T - T_{ref})]}$$

Note on ΔH_{rxn} : In derivation, X is of limiting reactant, so we wrote reaction as:



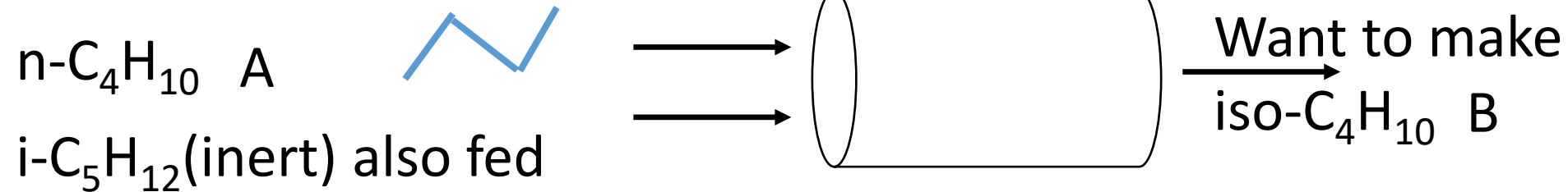
Then we can solve this along with the mole balance equation if we know something about the

- rate law (e.g., $r = kC_A^\alpha$)
- rate law parameter dependence on T (k , K_C)
- stoichiometry

Example problem: Isomerization of n-C₄H₁₀ (butane)

Example 11-3 Fogler

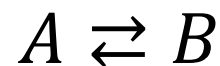
Liquid phase, elevated pressure



$$F_{T0} = 163 \text{ kmol/hr}$$

T₀ = 330 K, but NOT isothermal! Adiabatic reactor

90% nC4, 10% i-C5



$$k_f(360K) = 31.1 \text{ 1/hr}$$

$$\Delta H_{\text{rxn}} = -6900 \text{ J/mol C4, } E_a = 65.7 \text{ kJ/mol; } K_C = 3.03 @ 60^\circ \text{C;}$$

$$C_{A0} = 9.3 \text{ kmol/m}^3$$

$$C_{P,n\text{-C4}}^A = C_{P,i\text{-C4}}^B = 141 \text{ J/mol}\cdot\text{K; } C_{P,i\text{-C5}}^{\text{inert}} = 161 \text{ J/mol}\cdot\text{K}$$

From our mole balance algorithm:

$$15.77 m^3/hr \int_0^X \frac{1}{k_f \left((1 - X) - \frac{X}{K_C} \right)} dX = V_{PFR}$$

Recall we CANNOT solve this just yet, because some of these parameters in the integral are a function of X!

Using our adiabatic energy balance equation we got (for this particular reactor):

$$T = (43.4 \text{ K})X + 330 \text{ K}$$

May call it X_{EB} or $X_{adiabatic}$ to indicate its from the EB

$$k_f(T) = k_f(360\text{ K}) \exp \left[-\frac{E_a}{R} \left(\frac{1}{T} - \frac{1}{360\text{ K}} \right) \right]$$

$$= 31.1\text{ hr}^{-1} \exp \left[-\frac{65700\text{ J/mol}}{8.314\text{ J/mol} \cdot \text{K}} \left(\frac{1}{T} - \frac{1}{360\text{ K}} \right) \right]$$

$$K_C(T) = K_C(333\text{ K}) \exp \left[-\frac{\Delta H_{rxn}}{R} \left(\frac{1}{T} - \frac{1}{T_1} \right) \right]$$

$$= 3.03 \exp \left[-\frac{-6900\text{ J/mol}}{8.314\text{ J/mol} \cdot \text{K}} \left(\frac{1}{T} - \frac{1}{333\text{ K}} \right) \right]$$

We now can solve our mole balance equation because we know how k_f and K_C are related to T , and how T is related to X

Here solving/plotting in Mathematica, but full problem solved on page 563 in book using Polymath (full code on Canvas)

(* Define k_f and K_c *)

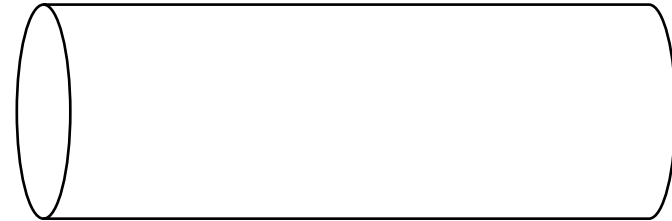
$k_f[T_] := 31.1 * \text{Exp}[-65700 / 8.314 * (1 / T - 1 / 360)]$

$K_c[T_] := 3.038 \text{Exp}[6900 / 8.314 * (1 / T - 1 / 333)]$

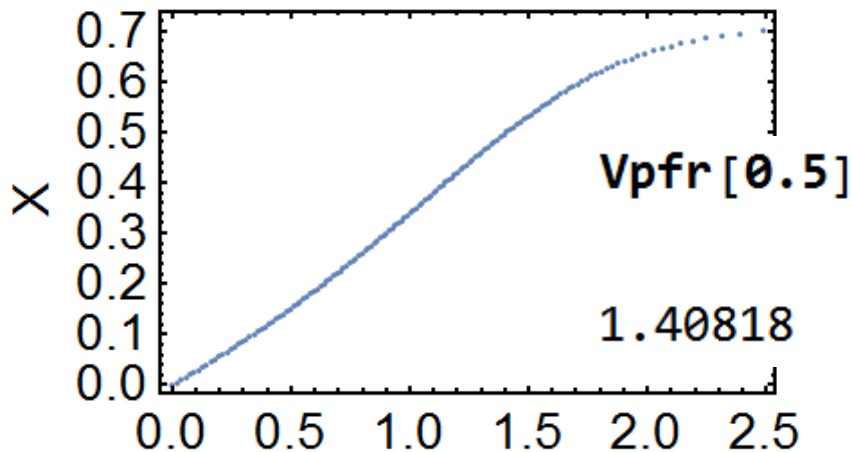
(* $T = f(X)$ *)

$T[x_] := 43.3 * x + 330$

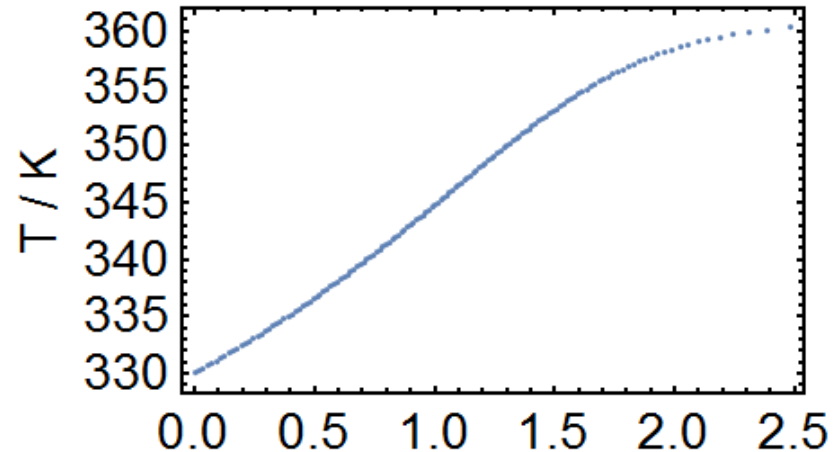
$\text{Vpfr}[xx_] := 15.77 * \text{NIntegrate}[1 / (k_f[T[x]] * ((1 - x) - x / K_c[T[x]])), \{x, 0, xx\}]$



Adiabatic PFR



Adiabatic PFR



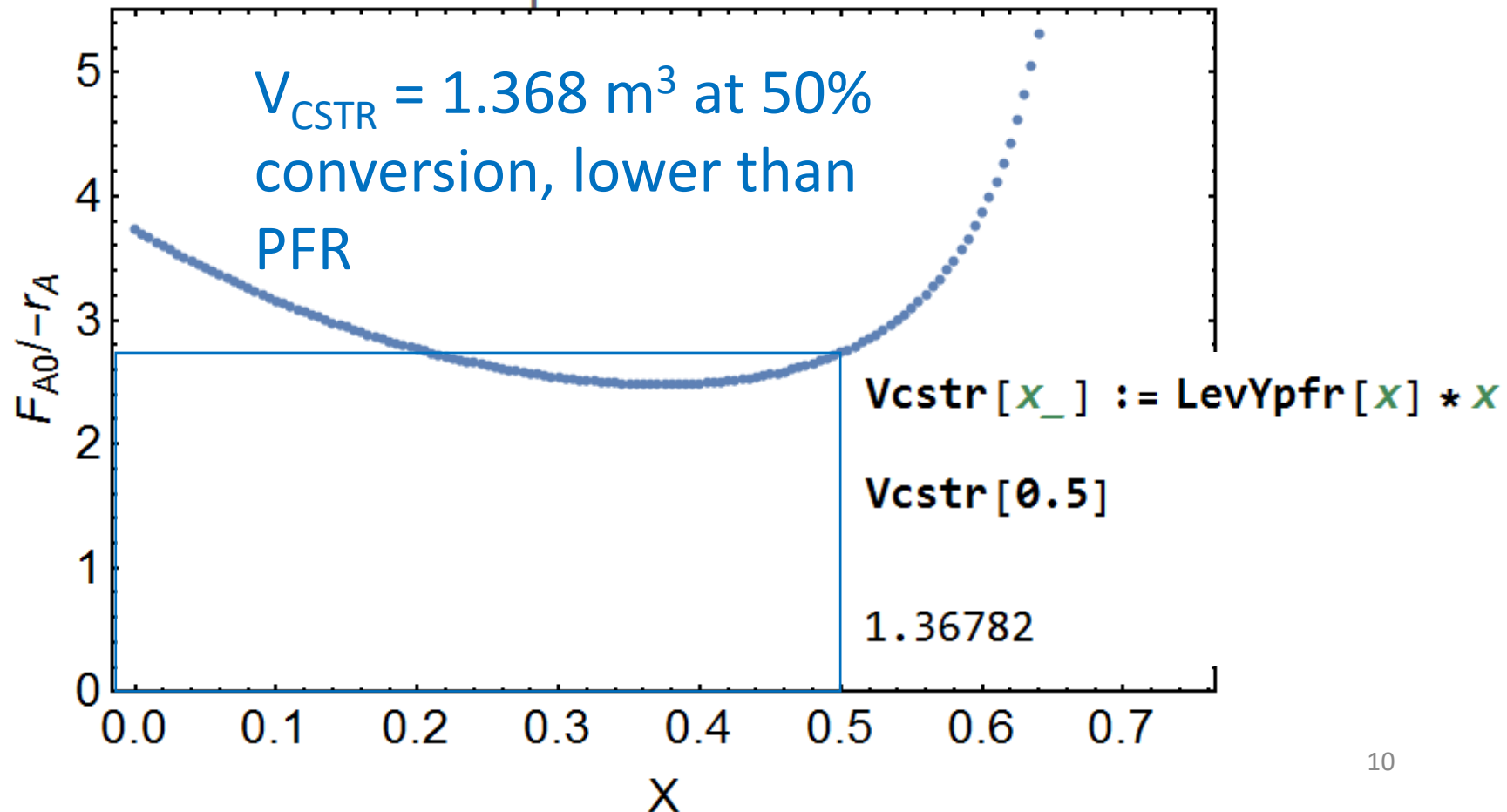
V / m^3 $V_{\text{PFR}} = 1.408 \text{ m}^3$ at $X = 50\%$ V / m^3

Levenspiel Plot

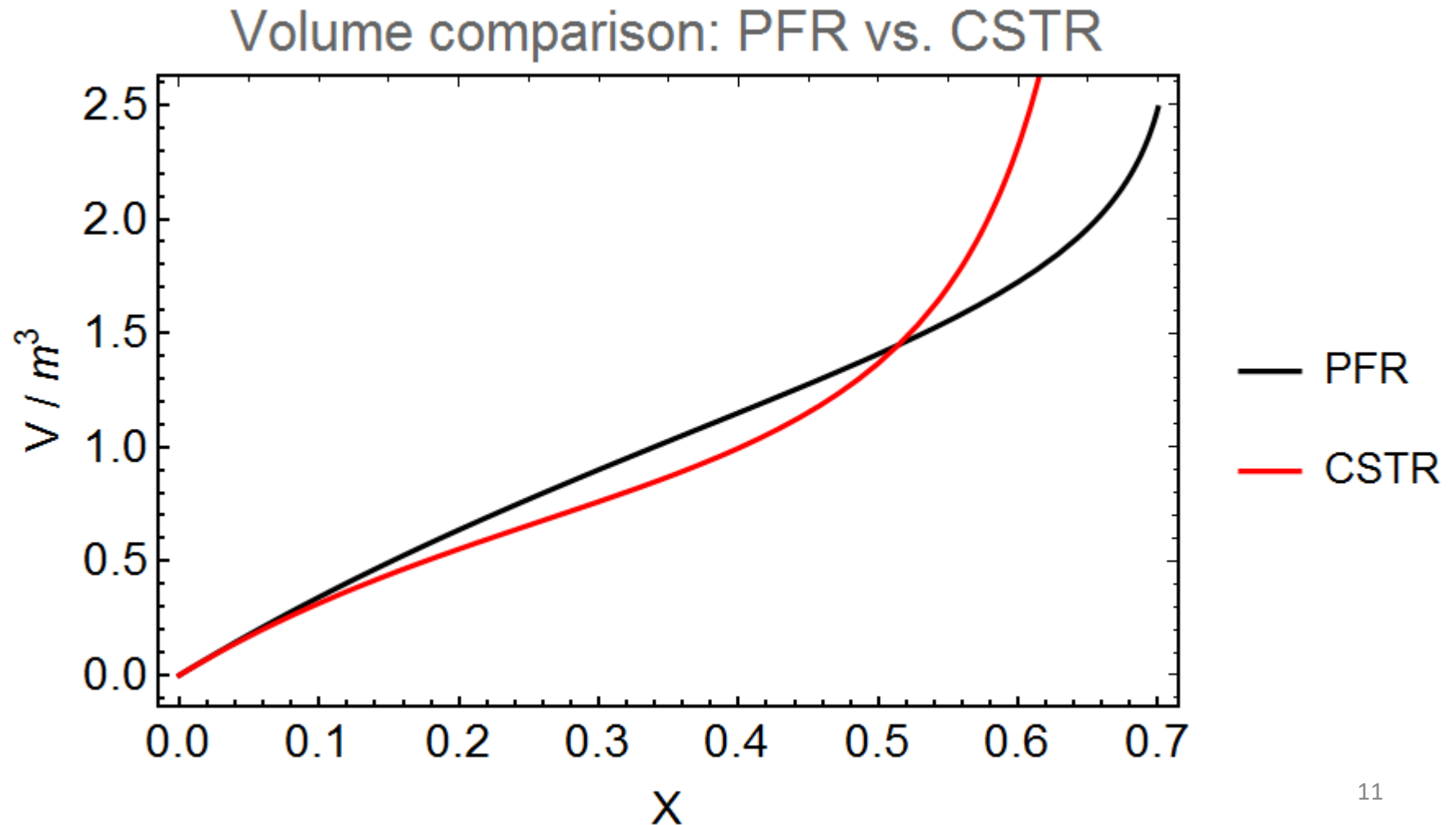
(* Want to plot $F_{A0}/-r_A$ as the y axis,
first define it *)

```
LevYpfr[x_] := 15.77 / (kf[T[x]] * ((1 - x) - x / Kc[T[x]]))
```

Levenspiel Plot: Adiabatic

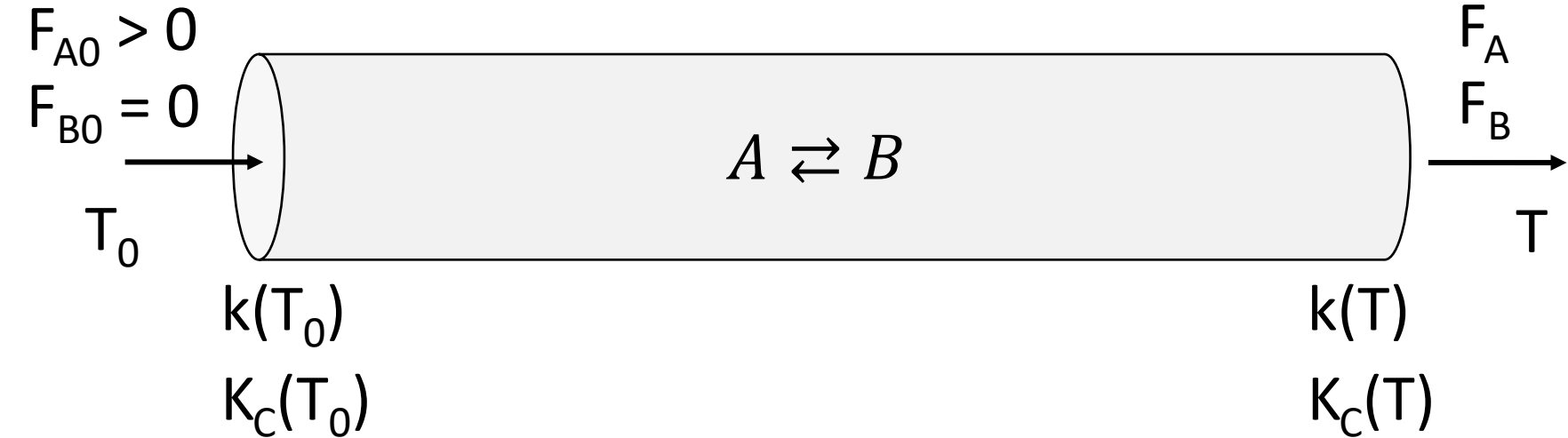


Plotting the V_{CSTR} and V_{PFR} functions together to compare volumes at different conversions



Discuss with your neighbors:

For an adiabatic reactor, if $T > T_0$, which is true?



A) $k(T) > k(T_0)$; $K_C(T) < K_C(T_0)$

Exothermic:

$X > 0, T > T_0, \Delta H_{rxn} < 0$

B) $k(T) > k(T_0)$; Cannot tell about $K_C(T_0)$ vs. $K_C(T)$

C) $k(T) > k(T_0)$; $K_C(T) > K_C(T_0)$

D) $k(T) < k(T_0)$

$$K_C(T) = K_C(T_0) \exp \left[-\frac{\Delta H_{rxn}}{R} \left(\frac{1}{T} - \frac{1}{T_0} \right) \right]$$

$$A \rightleftharpoons B$$

At equilibrium, no net reaction

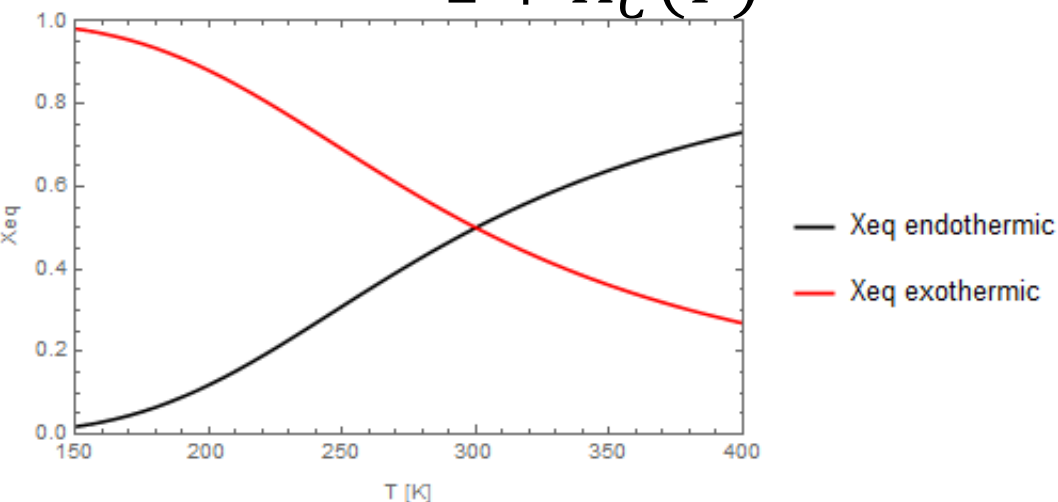
$$k_f C_{A,eq} = k_r C_{B,eq} \quad K_C = \frac{k_f}{k_r}$$

$$C_A = C_{A0}(1 - X)$$

$$C_B = C_{A0}(X)$$

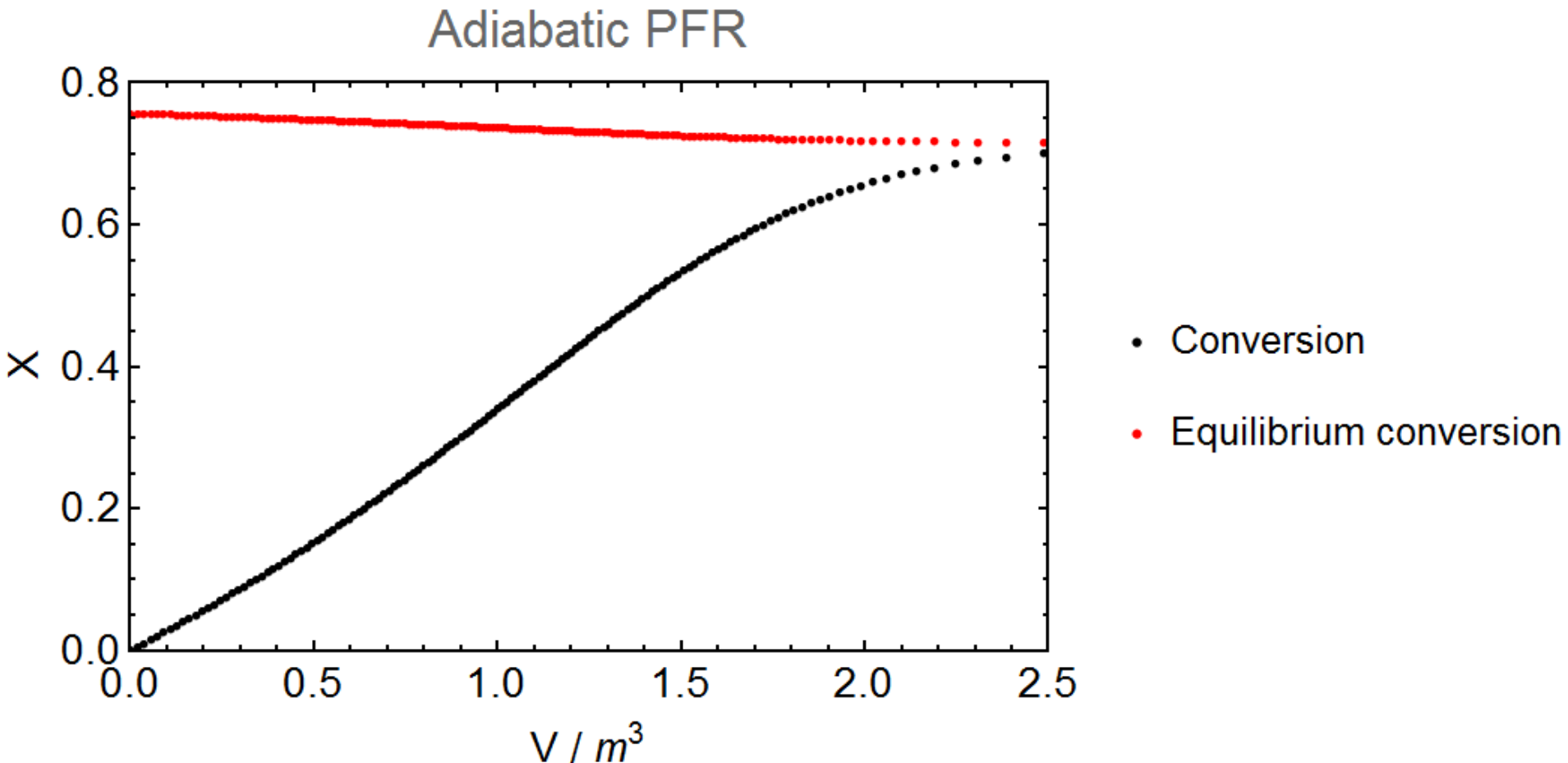
$$K_C C_{A0}(1 - X_{eq}) = C_{A0} X_{eq}$$

$$X_{eq}(T) = \frac{K_C(T)}{1 + K_C(T)}$$



Taking products out or cofeeding will change X_{eq} .
 But also, we could control the T!

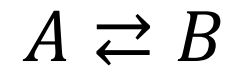
Plotting the equilibrium vs. reactor conversion as a function of reactor volume for our butane reactor.



(* What is the equilibrium conversion through the reactor *)

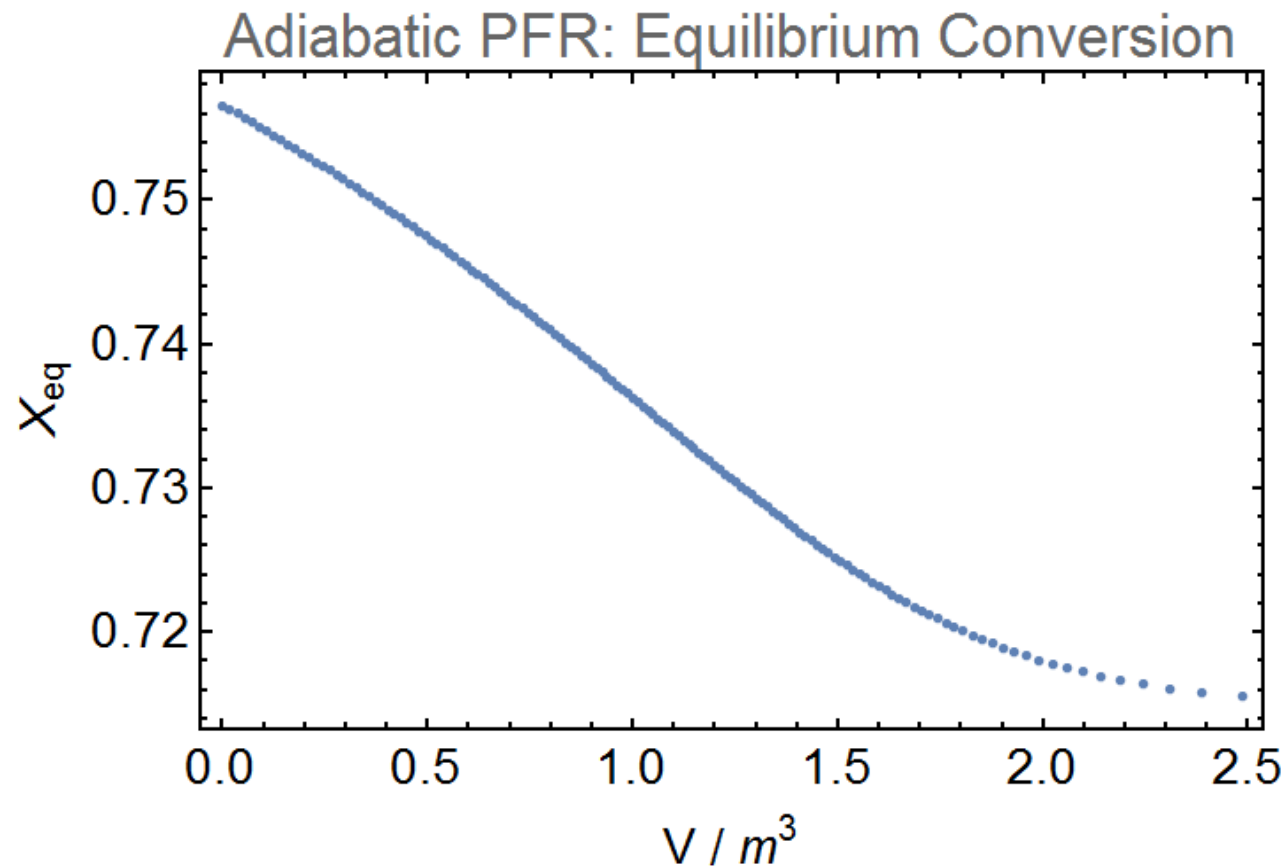
```
ListPlot[Table[{Vpfr[x], Kc[T[x]] / (1 + Kc[T[x]])}, {x, 0, 0.7, 0.005}],  
FrameStyle → Directive[Black, FontColor → Black], BaseStyle → {FontSize → 14},  
Frame → True, FrameLabel → {"V / m3", "Xeq"},  
PlotLabel → "Adiabatic PFR: Equilibrium Conversion"]
```

For the reaction:



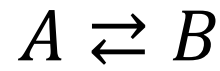
$$K_C = \frac{C_{B,eq}}{C_{A,eq}}$$

$$X_{eq} = \frac{K_C}{1 + K_C}$$

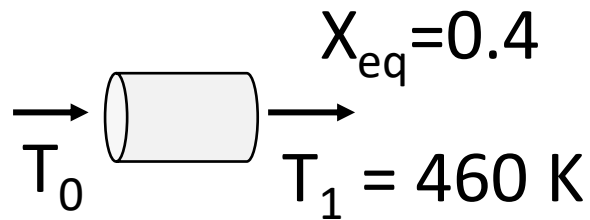


Continuing adiabatic reactor design

Exothermic

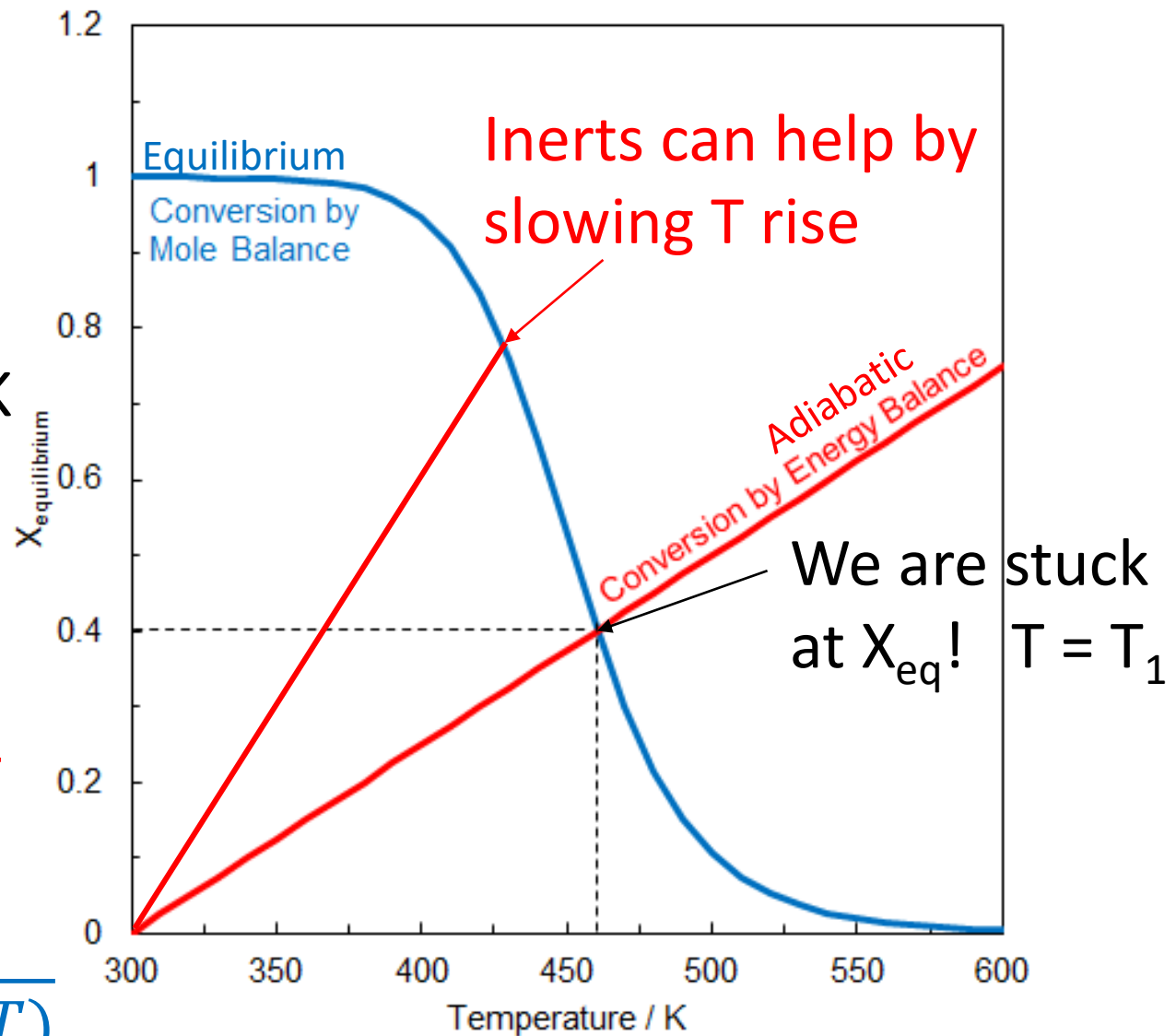


Single adiabatic reactor

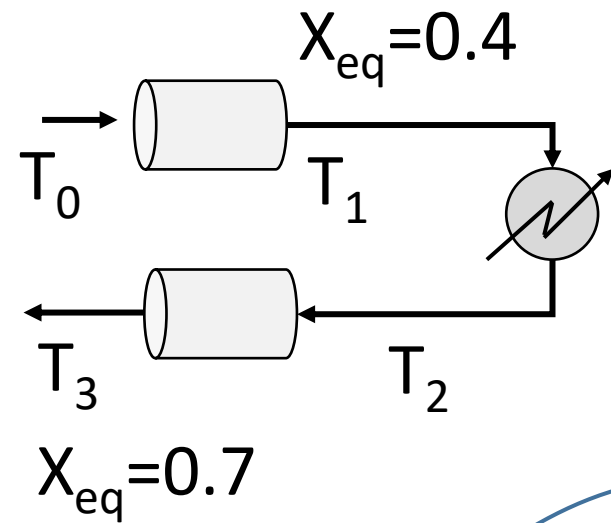
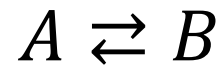


$$X_{adiabatic,EB} = \frac{\sum \theta_i C_{P,i} [T - T_0]}{-[\Delta H_{rxn}(T)]}$$

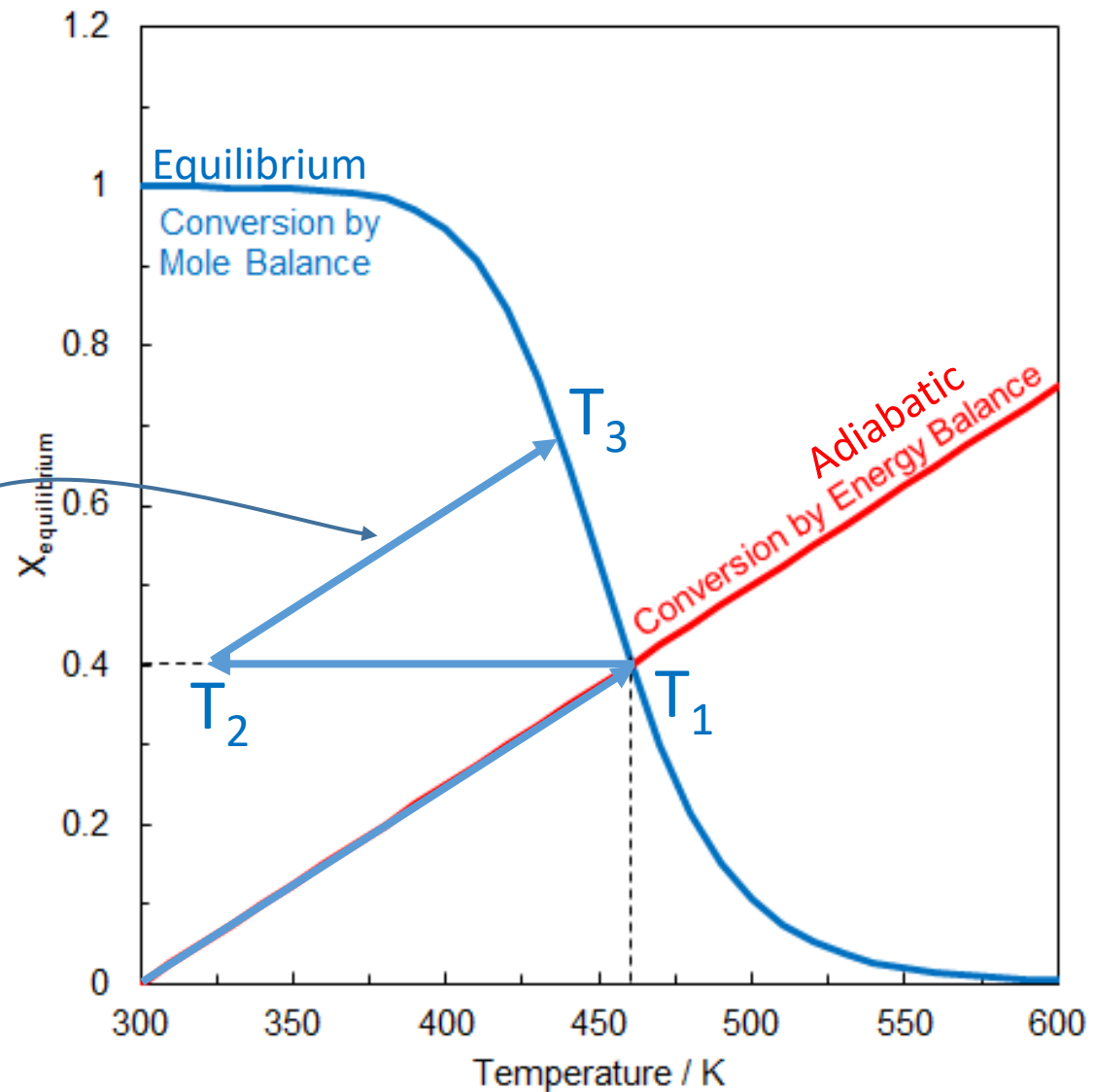
$$X_{eq}(T) = \frac{K_C(T)}{1 + K_C(T)}$$



What if we cooled in between reactors?



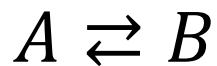
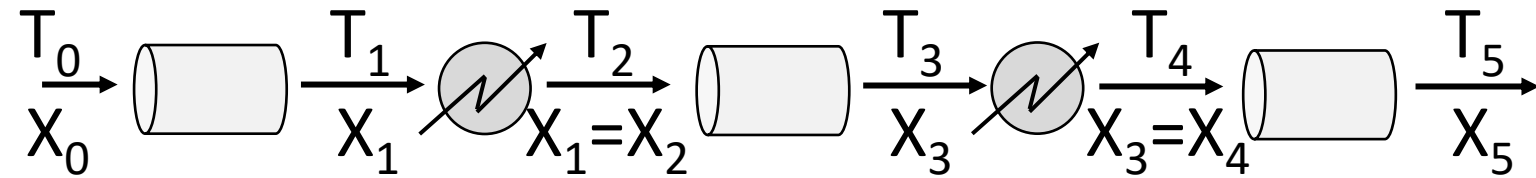
Same slope, but
different intercept
(T_0, X_0)



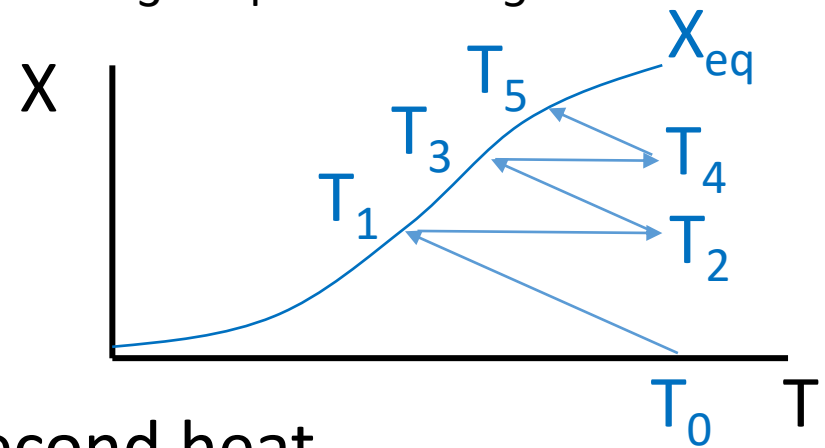
Interstage cooling

Discuss with your neighbors:

For a series of adiabatic reactors, what should the interstage heat exchangers do to the fluid to maximize X?



Endothermic



- A) First heat exchanger cool, second heat
- B) First heat exchanger heat, second cool
- ☒ C) Both heat
- D) Both cool

What if we want to control the temperature of our reactor?

We may want to do this for:

Safety

What if the reactor gets too hot?

<http://umich.edu/~safeche/>

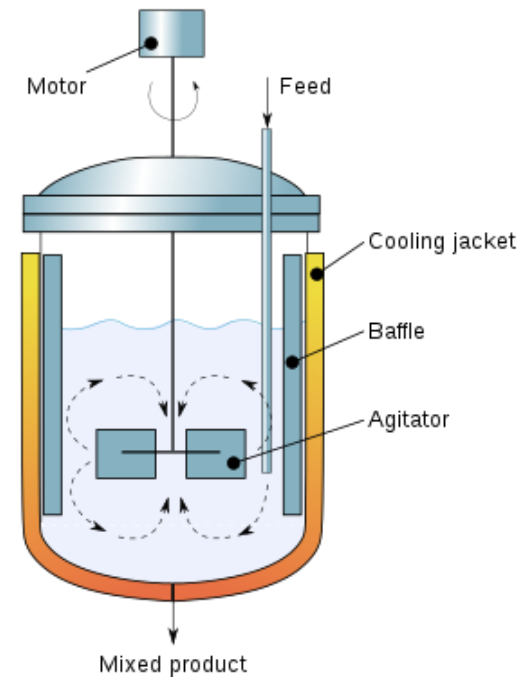
Control equilibrium/selectivity/conversion

Keep your reactor working

- Polymer production
- Catalytic regeneration (avoid sintering)

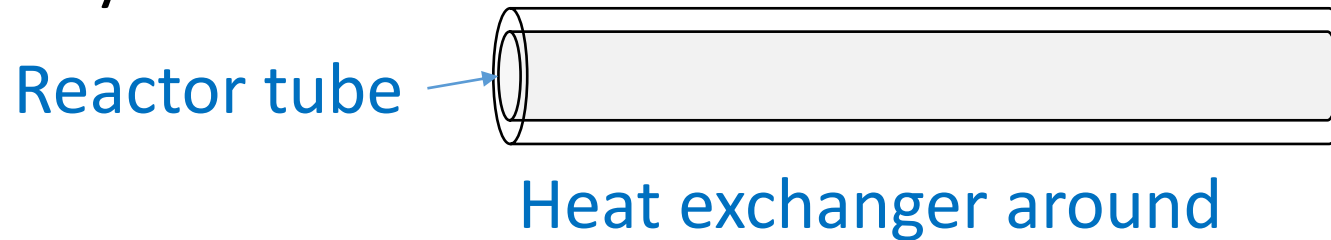
Easier to model/take rate data

Isothermal reactors much easier to deal with!



Often, we will control the temperature with heat exchangers

We used these for interstage (in between reactors), but we can also have a heat exchanger integrated with our reactor PFR today



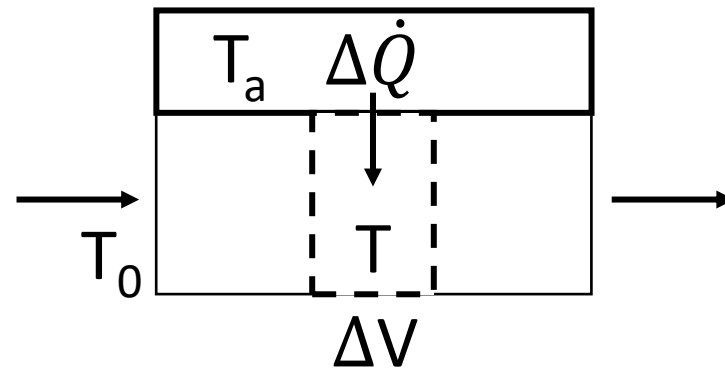
There will be an additional heat term that we previously did not include for adiabatic reactors

Eventually...

- We will need an energy balance on heat exchanging fluid and reactor
- For CSTRs, we will find that there are multiple steady states
- Dealing with multiple reactions each with a ΔH_{rxn}

Industrial reactors will generally have heat exchangers (no longer adiabatic): Recall our energy balance

$$F_{A0} \left[\left(\sum -\theta_i C_{P,i} [T - T_0] \right) - [\Delta H_{rxn}(T_{ref}) + \Delta C_P (T - T_{ref})] X \right] + \dot{Q} - \cancel{\dot{W}_{shaft}^0} = \frac{d\hat{E}_{sys}^0}{dt}$$



No mass transport through walls, only heat

Reactor with coolant/heating **adjacent to reactant volume**.
 $\dot{Q} \neq 0$. In reactant slug have a $\Delta \dot{Q}$ term from V to $V + \Delta V$.
 T_a is the cooling/heating material temp.

How does heat transfer?

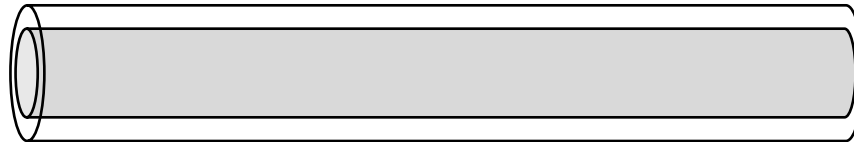
Need a **temperature gradient**

$$\Delta \dot{Q} = U \Delta A (T_a - T)$$

U is the heat transfer coefficient with units of $\text{J}/\text{m}^2 \cdot \text{s} \cdot \text{K}$

ΔA is the heat transfer area with respect to ΔV

$$\Delta A \equiv \underbrace{a}_{\substack{\text{shape factor,} \\ \frac{\text{heat transfer area}}{\text{unit reactor volume}}}} \Delta V$$



For example, if heat exchanger surrounds a tubular reactor:
a very small diameter tube reactor (**large a**)

vs.
a very large diameter tube (**small a**)

$$a = \frac{\pi \Delta L D}{\frac{\pi}{4} \Delta L D^2}$$

Energy balance on a reactor element ΔV

In – out + gen – cons = accumulation = 0

$$\sum F_i H_i \Big|_V - \sum F_i H_i \Big|_{V+\Delta V} + Ua\Delta V(T_a - T) = 0$$

This is the balance on the reactor volume ΔV . This time we have a term outside the sums that is non-zero, that has ΔV

$$\frac{\sum F_i H_i \Big|_V - \sum F_i H_i \Big|_{V+\Delta V}}{\Delta V} + Ua(T_a - T) = 0$$

Limit as ΔV goes to zero,

$$-\frac{d[\sum F_i H_i]}{dV} + Ua(T_a - T) = 0$$

Product rule of derivatives

$$Ua(T_a - T) = \frac{d[\sum F_i H_i]}{dV} = \sum F_i \frac{dH_i}{dV} + \sum H_i \frac{dF_i}{dV}$$

$$H_i(T) = H_i^0(T_{ref}) + \int_{T_{ref}}^T C_{P,i} dT$$

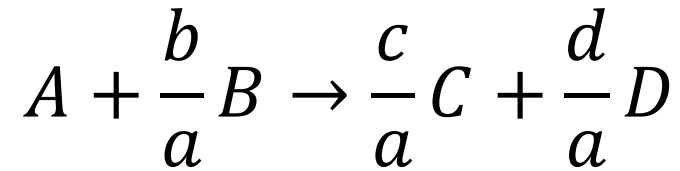
$$\frac{dH_i}{dV} = \frac{dH_i}{dT} \frac{dT}{dV} = C_{P,i} \frac{dT}{dV}$$

To evaluate the $\frac{dF_i}{dV}$ terms, we will need to consider more details about the reactor.

This is different than for our adiabatic case where we had a general equation that applied to several different reactor types. (Adiabatic had same EB for CSTR/PFR)

For PFR we get relation between flow rate and volume from our mole balance:

$$\frac{dF_i}{dV} = r_i$$



$$\frac{dF_A}{dV} = r_A;$$

$$\frac{dF_B}{dV} = r_B = \frac{b}{a}r_A; \frac{dF_C}{dV} = r_C = -\frac{c}{a}r_A; \frac{dF_D}{dV} = r_D = -\frac{d}{a}r_A$$

$$\frac{dF_i}{dV} = -\nu_i r_A$$

Plugging back into our energy balance for a PFR

$$Ua(T_a - T) = \sum F_i \left(C_{P,i} \frac{dT}{dV} \right) + \sum H_i (-v_i r_A)$$

$$Ua(T_a - T) = \sum F_i C_{P,i} \frac{dT}{dV} - \sum H_i v_i r_A$$

$$Ua(T_a - T) = \frac{dT}{dV} \sum [F_i C_{P,i}] - r_A \Delta H_{rxn}$$

-Heat 'removed' Heat 'generated'

$$\frac{dT}{dV} = \frac{Ua(T_a - T) + r_A \Delta H_{rxn}}{\sum F_i C_{P,i}}$$

$$F_i = F_{A0}(\theta_i + v_i X)$$

PFR mole balance design equation:

$$F_{A0} \frac{dX}{dV} = -r_A$$

PFR energy balance with heat exchanger:

$$\frac{dT}{dV} = \frac{Ua(T_a - T) + r_A \Delta H_{rxn}}{\sum F_i C_{P,i}} = \frac{r_A \Delta H_{rxn} - Ua(T - T_a)}{F_{A0} [\sum \theta_i C_{P,i} + \Delta C_P X]}$$


$$F_i = F_{A0} (\theta_i + \nu_i X)$$

Now we have two non-linear coupled ODEs, can solve with software of our choice if we assume T_a is constant everywhere in the reactor.

ΔH_{rxn} at T , $r_A \Delta H_{rxn}$ is positive (negatives cancel) for proceeding exothermic reaction

For an exothermic reaction in a PFR, how much total heat must be removed to maintain isothermal operation? $A \rightarrow B$

$F_{A0} = 1 \text{ mol min}^{-1}$
 $T_0 = 300 \text{ K}$
 $\Delta H_{rxn} = -20 \text{ kJ mol}^{-1}$


$$F_{A0} \frac{dX}{dV} = -r_A$$
$$\frac{dT}{dV} = \frac{r_A \Delta H_{rxn} - Ua(T - T_a)}{\sum F_i C_{P,i}}$$

total heat removed $= \int Ua(T - T_a) dV$

For $dT/dV = 0$ $F_{A0} \frac{dX}{dV} = \frac{-Ua(T - T_a)}{\Delta H_{rxn}}$

Total $Q_{removed} = -1 \text{ mol min}^{-1} (0.5) (-20 \text{ kJ mol}^{-1})$

Total $Q_{removed} = 10 \text{ kJ min}^{-1}$

If isothermal, E.B. is

$$\frac{dT}{dV} = 0 = \frac{r_A \Delta H_{rxn} - Ua(T - T_a)}{\sum F_i C_{P,i}}$$

$$0 = r_A \Delta H_{rxn} - Ua(T - T_a)$$

$$r_A = \frac{Ua(T - T_a)}{\Delta H_{rxn}}$$

Mole balance is

$$F_{A0} \frac{dX}{dV} = -r_A = -\frac{Ua(T - T_a)}{\Delta H_{rxn}}$$

Rearrange, then take integral

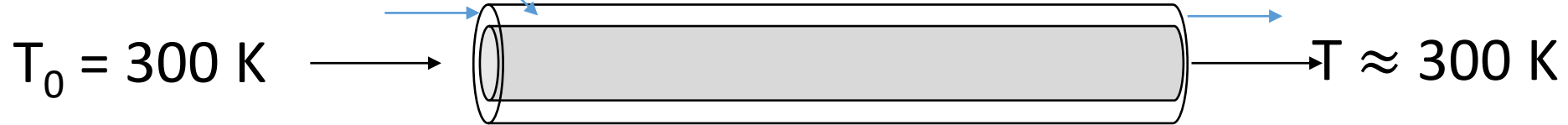
$$F_{A0} \Delta H_{rxn} dX = -Ua(T - T_a) dV$$

$$\int F_{A0} \Delta H_{rxn} dX = - \int Ua(T - T_a) dV$$

$$\text{Total rem.} = \int Ua(T - T_a) dV = -1 \text{ mol min}^{-1} (0.5) (-20 \text{ kJ mol}^{-1}) = 10 \text{ kJ min}^{-1}$$

Where would this heat be coming from? From the heating fluid, which may be flowing co or counter-current

$$T_a = 280 \text{ K}$$



Here co-current

In reality, if the heating fluid is removing heat, it itself must be heating up, so $T_a = f(V)$

If T_a is not constant, we will need an energy balance on the heating/cooling element (next Tuesday)